

US 4,166,711 and US 2,303,487 show further fastening types for exchangeable tool heads.

US 2,164,573 shows thin reaming heads, which are punched from sheet metal or may be
5 sawn from a toothed rod. This necessitates the use of comparatively soft steels. A flexible steel is also necessary for one of the embodiments, with which the teeth are bent on machining. For this reason, one may only expect a limited accuracy of machining, particularly with the disclosed, relatively thin teeth.

10 A flat reaming head is described in US 2,164,571, which may be operated in both rotational directions. The reaming head may also be turned over and be used with the same rotational direction of the shaft. For this however, a large number of small teeth lying close to one another must be present, and the teeth must be ground equally on both sides along the
15 periphery of the interchangeable head. The individual teeth in each case therefore are symmetrical and comprise two cutting edges, one for each rotational direction. The geometry of the cutting edges - with a negative rake angle - does not however permit any reaming, but a scraping machining at best.

US 5,163,790 shows a conical mounting, wherein the conical projection comprises three
20 cutouts. On pressing together, the conical projection is pressed apart at the locations which are not relieved and contracts at the cutouts. A fixation of a shaft in a region above the cone connection is achieved by way of this.

US 3,087,360 describes a movable connection between the shaft and reaming head,
25 which compensates positioning inaccuracies.

DESCRIPTION OF THE INVENTION

30 It is the object of the invention to provide a machine reaming tool, an interchangeable head and a shaft for a machine reaming tool of the initially mentioned type, which permits a material saving and thus an inexpensive manufacture of reaming heads. A further object of the invention is to permit a simplified design of the mentioned parts of a machine reaming tool.

35 These objects are achieved by the subject-matters of the respective independent patent claims.

The machine reaming tool according to the invention comprises a reaming head or an interchangeable head and a shaft according to the invention. The interchangeable head according
40 to the invention thereby is of one piece and exchangeable, wherein, in the axial direction, it has a

thickness of less than h_{\max} millimetres at every location, thus including a means for exchange adaptation, wherein this thickness h_{\max} is computed from a diameter D1 of the interchangeable head in millimetres as

5
$$h_{\max} = 6\text{mm} + 1/10(D1 - 12\text{ mm}).$$

In a preferred embodiment of the invention, h_{\max} (for D1 smaller than 12 mm) is at least 6 mm and (for D1 larger than 35 mm) is 8 mm at the most. In a preferred embodiment, h_{\max} is equal to 6 mm independently of the diameter D1.

10 Accordingly, a guide portion on individual cutters of the interchangeable head is also somewhat shorter than h_{\max} . This design of the interchangeable head is based in the recognition that a common, comparatively long guide portion of the cutter is not necessary at all in many cases of application. By way of this, the interchangeable head, may be manufactured in a material-saving manner, even with smaller thickness, for example below 5 mm, 4.5 mm, 4 mm,
15 3.5 mm, 3 mm, 2.5 mm or 2 mm.

In a preferred embodiment of the invention, the interchangeable head, as a connection element, comprises a cutout in a plane end-face on the shaft side, for the centring fastening on
20 the shaft. The shaft has no machining means such as teeth or cutters, and has a larger outer diameter than the interchangeable head. The shaft is preferably of one piece, and on an end-side plane surface comprises a connection lug which projects out of this plane surface in the axial direction and which corresponds to the cutout of the interchangeable head. The interchangeable head thus has no axially projecting lug for exchange adaptation. The material requirement on the
25 interchangeable head is further reduced, and the manufacture of the interchangeable head is simplified by way of this.

With the conventional connection type, the projecting lug of the hard interchangeable head expands the comparatively soft shaft of the base body, wherein this expansion is favoured
30 by a separating gap. In contrast to this, according to the invention, an inner surface or several points of the cutout of the hard interchangeable head press the projecting lug of the shaft together. This compression of the shaft material is effected with an axially centred arrangement of the connection elements in essentially the radial direction. The compression, with regard to the material displacement, requires comparatively higher forces than the expansion of the shaft
35 material according to the state of the art. Accordingly, according to the invention, greater manufacturing accuracies are required as the case may be. Advantageously, in contrast to the expansion, the planarity of the end-side surface of the shaft is not negatively influenced.

In other words therefore, the interchangeable head comprises two parallel, plane end-
40 faces with an axial distance of less than approx. 6, preferably less than 5 or less than or equal to 4

millimetres, and without an exchange adaptation projecting beyond these end-faces. A plane end-face on the shaft side comprises a recess or sinking which projects into the end-face, and into which a corresponding, projecting part of the shaft may be inserted. With this insertion, thus when the interchangeable head is pressed against the shaft in the axial direction, this projection is pressed together or compressed or inwardly deformed along its complete periphery or at least three locations, by way of the interchangeable head.

More than one centring recess in the interchangeable head and accordingly more than one lug on the shaft is possible in other embodiments.

In a preferred embodiment of the invention, the cutout in the end-face of the interchangeable head is a conical socket and accordingly the projecting lug on the shaft is a corresponding conical projection. The dimensions of the conical socket and conical projection are preferably matched to one another such that on placing the interchangeable head onto the shaft, the end-face plane surface of the shaft and the oppositely lying end-side of the interchangeable head have a predefined distance. For fastening the interchangeable head, at least one cap screw is guided through at least one bore in the interchangeable head, and is screwed in the shaft. On tightening the cap screw or the head screws, the conical projection of the shaft is pressed together by the conical socket of the hard interchangeable head, until the end-side plane surface of the shaft and the oppositely lying end-side of the interchangeable head meet one another. A frictional connection arises on account of the pressing-together of these two end-faces. It has been shown that this connection is sufficient for the transmission of torque in normal operation, even with the use of a single, axially centred cap screw. Thus a pure cone connection may be used, without polygonal force transmission means such as triangular or hexagonal connections for example. The manufacture of the interchangeable head as well as of the shaft is simplified by way of this.

The conical projection or the conical socket comprise several, preferably three slightly projecting or exposed segments. When the reaming tool is assembled, these form contact surfaces between the shaft and the interchangeable head, and thus a three-point contact for example. On manufacture, only these segments need to be manufactured with a high accuracy, for example ground, and not the complete conical periphery.

In another embodiment of the invention, the cutout in the end-face of the interchangeable head, which cutout is designed as a connection element, is essentially circularly cylindrical and at three locations of the inner cylinder periphery in each case comprises a contact segment or a contact point, at which the cutout is designed somewhat more narrowly. In the peripheral direction, these locations are distributed distanced from one another over the periphery. Accordingly, the projecting lug designed as a connection element on the shaft is likewise essentially circularly cylindrical.

In a preferred variant of this embodiment, the contact segments are formed by plane surfaces. An extension of one of the contact segments in the peripheral direction is comparatively larger than the extension of the two other ones, preferably one and a half to twice as large. By way of this, this contact segment acts as a catch segment. In a manner corresponding to this, the shaft on a part segment of the periphery comprises a plane surface, said part segment or sector encompassing for example up to one eighth of the periphery. The catch segment also acts as a rotational securement. The orientation of the interchangeable head on placing on the shaft is fixed in an unambiguous manner, since the catch segment must be accordingly orientated to the plane surface on the shaft. Furthermore, the catch segment effects a positive-fit force transition onto the interchangeable head, and as the case may be, a locking of the interchangeable head and shaft on rotation. The interchangeable head, analogously to that described above, is screwed to the shaft by way of one or more preferably sunk screws, and pressed onto the end-side of the shaft.

The cylindrical shape of the connection elements on the interchangeable head and shaft permit a simple and inexpensive manufacture, with a sufficient exchange accuracy of the connection.

In further embodiments of the invention, the connection elements in the interchangeable head and shaft are rounded polygon cylinders, preferably triangular cylinders which are rounded along the periphery. The outer cylinder on the shaft is slightly smaller than the inner cylinder on the interchangeable head, so that three contact points form on the periphery of the cylinder when attaching the interchangeable head onto the shaft and a small mutual rotation. The polygon cylinders are either shaped asymmetrically, or the polygon cylinders, as described below, on the end-faces to be pressed onto one another, comprise corresponding recesses and projecting elements, as an orientation means for orientation of the interchangeable head in the peripheral direction.

The centring cutout in the interchangeable head with all embodiments, is not compellingly central and continuous. With the use of exactly one cap screw however, the cutout is arranged centrally or axially centrally in the interchangeable head and forms a through hole. The cutout on the end-face of the interchangeable head which is distant to the shaft, then preferably forms a cutout for screw head. The screw head may be sunk in the interchangeable head by way of this. In turn, the machining depth in a pocket hole may be maximised by way of this.

In a further preferred embodiment of the invention, the shaft and the interchangeable head comprise corresponding means for the unambiguous orientation of the interchangeable head with respect to the shaft in the peripheral direction. The orientation means on the

interchangeable head for example is a sinking or a hole in the end-face facing the shaft, and the corresponding orientation means on the shaft is a projection or a pin. These orientation means ensure that the high demands placed on the truth of running characteristics of the reamer are also met on exchange of the interchangeable head. The reaming heads on manufacture, are ground on
5 the same machine tool, wherein the orientation of the reaming heads with respect to the orientation means is the same in each case. A machine which applies the reaming heads, when setting up with a new reamer, must be trued concentrically in the micrometer range. If then the interchangeable head of the reamers is exchanged, then no renewed concentrically trueing is necessary thanks to the orientation means. The orientation means also contribute to the force
10 transmission to the interchangeable head, on account of their positive fit.

The interchangeable head along its peripheral direction comprises several cutters or cutting teeth, which are distanced to one another. Each cutter comprises a leading cut portion and a guide portion. A leading cut portion in the axial direction preferably has a length of 0.03 mm to
15 1.2 mm, in particular of approx. 0.3 to 0.7 mm. The length of the cutter remaining up to the thickness of the interchangeable head forms the guide portion. A ratio between the length of the leading cut portion and the length of the guide portion of 1:7 results for an average length of the leading cut portion of 0.5 mm and an interchangeable head thickness of 4 mm. This ratio is preferably between 1:6 and 1:10, which are comparatively large values compared to the maximal
20 usual values of 1:20 for example. The large values result from the low thickness of the interchangeable head. This in turn is possible amongst other things thanks to the recognition that the greater part of all applications does not place particularly exacting demands on the guiding of the reamers.

25 The point between the leading cut portion and the transition region is hereinafter called the effective cutting corner or reaming corner. Proceeding from this point, the guide region tapers slightly to the rear, so that the reaming corner is the point of the cutter with the largest radius, and thus also determines the radius or diameter of the machined hole. Here and in the following, "front" in each case indicates the side of the interchangeable head which is distant to
30 the shaft, and "rear" indicates the opposite side.

In a preferred embodiment of the invention, an interchangeable head is designed as an reversible (or indexable) insert. The interchangeable head thus comprises a first side and an oppositely lying second side, and may be selectively assembled with the first or the second side
35 against the shaft, and may be used for reaming in both cases. The first or the second side, for example with a plane surface and a centring cone, are held or pressed against the corresponding surfaces of the shaft by way of connection means such as screws or tie rods.

40 With this, it is possible in a first phase to firstly assemble and use the interchangeable head with the one side as the rear side against the shaft, until the cutters, in particular the reaming

corners, are worn on the front side. Subsequently, the interchangeable head is indexed for a second phase, which means the other side becomes the rear side and the leading cut portion and reaming corners which hitherto were located at the rear, then come to the front side. The rotational direction must of course be reversed during the machining for this. Surprisingly, no significant wearing of the rear reaming corners takes place in the first phase, although these at first have the same radius as the front reaming corners, and after a certain wearing of the front reaming corners, even have a larger radius.

Thus in each case, it is the other side of a cutting edge of a tooth which is worn after reversing the interchangeable head. No second cutting edge per tooth is required in order to be able to operate with the reversible interchangeable head. The teeth are thus asymmetrically shaped along the periphery or with a direction viewed parallel to the rotational axis.

The reversible interchangeable head comprises centring means and fastening means for both fastening directions. For example, two coaxial conical sockets are present in place of the conical socket described above. Each of the conical sockets proceeds from one of the two plane surfaces and extends up to the middle of the interchangeable head, where the two conical recesses meet one another. As an alternative to this, the conical sockets do not extend to the middle, but are connected by a bore for a central fastening element.

In the case of a cylindrical instead of a conical recess, this is designed such that it has at least one mirror symmetry with respect to a plane perpendicular to the plane of the interchangeable head.

The fastening means are for example one or more continuous bores with cutouts on both sides for screw heads, or two separate sets of bores, wherein the one set has such cutouts on the one side of the interchangeable head, and the other set has cutouts on the other side. A design without cutouts is also conceivable.

The guide portion between the two reaming corners in each case of one cutter is preferably tapered towards the middle of the cutter. This taper may for example be V-shaped or circular-arc-shaped, corresponding to a hollow grinding. The tapering of the diameter towards the oppositely lying side is small and lies in the region of a hundredth of a millimetre per 10 mm in the axial direction. In a preferred embodiment of the invention, a groove or notch is arranged in the cutter roughly in the middle between the tapers. The manufacture of the two tapers which run towards one another, is simplified by way of this. What is surprising is the fact that an adequate guiding during the machining with the interchangeable head takes place even with such a short and possibly interrupted guide region.

In a further preferred embodiment of the invention, the thin design of the interchangeable head permits it to be combined with a second machining tool which is fastened on the same shaft. This second tool is a pre-machining tool for example, such as a planisher with a slightly smaller diameter than the interchangeable head. On application of such a combined tool, the planisher produces a first inner radius corresponding to a pre-machining diameter or predetermined size in a present bore, and the reaming head produces a slightly larger inner radius corresponding to the finished dimension of the bore, in one working procedure. The planisher and the interchangeable head are preferably fastened on the shaft with separate fastening means, or however by way of a single, common fastening means, a concentric screw for example. The shaft comprises corresponding fastening means for the interchangeable head and the second machining tool. The total height of the combined tool is small and the fact that this may be applied at all in restricted spatial conditions is due to the reduced thickness of the interchangeable head.

In yet a further preferred embodiment of the invention, which may likewise be utilised thanks to the slim design of the interchangeable head, the preferably single-piece interchangeable head is designed as a reaming tool as well as a milling tool. For this, in each case the front side of a cutter is primarily designed for reaming, as described above, and additionally the leading cut portion on the rear side is designed as a chamfer miller. The leading cut portion is preferably also designed larger on the front side than would be required merely for reaming, and on account of this, the front leading cut portion may also be used as a chamfer miller. A circular movement of the tool axis takes place (circular milling or interpolating milling) takes place on milling a chamfer at the entry or exit of a bore.

With the use of the last mentioned embodiment of the invention, a machining method is implemented with the following steps

- milling a chamfer at the entry of a bore by way of the leading cut portion on the front side of the cutters;
- reaming the bore to a nominal dimension by way of the leading cut portion and above all the reaming corner on the front side of the cutters;
- milling a chamfer at the exit of the bore by way of the leading cut portion on the rear side of the cutters; and
- leading back the tool through the bore, wherein any occurring burr material which has arisen with the previous step is removed.

Thanks to the combination of machining functions in a single, one-piece and thin interchangeable head according to the invention, one may implement the described machining steps also with restricted spatial conditions, without tool exchange and in one production step.

Various combinations of the embodiments described above are possible. The following are mentioned, which are not conclusive: The combination of chamfer milling and reaming may also be realised with an reversivble insert. One may also apply a cutter for chamfer milling on the rear side of an interchangeable head which is applied combined with a planisher.

5

The described interchangeable heads are preferably manufactured from a material manufactured by sintering, such as hard metal, cermet, ceramic (e.g. silicon ceramic or ceramic insert or oxide ceramic) or CBN (cubic boron nitride).

10

Further preferred embodiments are to be deduced from the dependent patent claims.

BRIEF DESCRIPTION OF THE DRAWINGS

15

The subject-matter of the invention is hereinafter described in more detail by way of preferred embodiments which are shown in the accompanying drawings. There are schematically shown in:

20

Figure 1 a longitudinal section through a head region of a machine reaming tool according to the invention;

Figure 2 a longitudinal section through a head region of a shaft of a machine reaming tool according to the invention;

25

Figure 3 a shaft according to the invention;

Figures 4, 5 and 6, in each case, corresponding views of interchangeable heads according to various embodiments of the invention;

30

Figures 7 and 8 sections through an interchangeable head in a further embodiment of the invention;

35

Figure 9 corresponding sections through a shaft according to the interchangeable head from Figures 7 and 8;

Figure 10 connection elements of the interchangeable head and shaft, in a further embodiment of the invention;

40

Figure 11 an interchangeable head designed as a reversible insert, in a partly sectioned lateral view and a detailed view;

Figure 12 a lateral view in a cross section and a plan view of a reversible insert according to Figure 11;

5 Figure 13 a pre-machining tool for placing on the interchangeable head;

Figure 14 a shaft with an interchangeable head and pre-machining tool;

10 Figure 15 an interchangeable head designed as a combined reaming head and miller, in a partly sectioned lateral view and a plan view; and

Figures 16 and 17 in each case, corresponding views of interchangeable heads according to further embodiments of the invention.

15 The reference numerals used in the drawings and their significance are summarily listed in the list of reference numerals. Basically, the same parts are provided with the same reference numerals in the figures.

20 WAYS OF CARRYING OUT THE INVENTION

Figure 1 shows a longitudinal section through a head region of a machine reaming tool according to the invention. An interchangeable head 1 is screwed onto a shaft 2 by way of a cap screw 3 running centrically in the axial direction. The cap screw 3 is sunk in a pocket 14. The interchangeable head 1 for orientating the interchangeable head 1 in the peripheral direction, comprises an orientation hole 12 and the shaft 2 comprises an orientation pin 22 which projects into the orientation hole 12. The shaft 2 comprises an axial bore 26 from which coolant channels 24 on the tool-side end lead to the outer side of the shaft 2 in the proximity of the interchangeable head 1. An improved cooling of the cutting parts is ensured on account of the proximity of the coolant exit to the interchangeable head 1 and the small thickness h of the interchangeable head. A shaft-side plane surface 15 of the interchangeable head 1 is pressed onto an end-side plane surface 25 of the shaft 2 by way of the screwing. Thereby, connection elements which in the present embodiment of the invention are a conical socket 11 of the interchangeable head 1 and a conical projection 21 of the shaft 2, are pressed onto one another. Since the material of the interchangeable head 1 has a lower deformability than that of the shaft 2, thereby the conical projection 21 is inwardly deformed in the radial direction within its material elasticity and is compressed or compacted.

40 The interchangeable head 1 has a diameter D_1 between for example 10 mm and 60 mm, wherein an embodiment with a fastening of the interchangeable head 1 according to Figure 5 is

preferred for higher values. The shaft 2 has a diameter D2 which is at least a few millimetres smaller than that of the interchangeable head 1.

A thickness h of the interchangeable head 1 in the axial direction, thus in the direction of the rotational axis of the tool is preferably for example less than 6 mm or 5 mm, in the present example is 4 mm, with a diameter D1 between 10 mm and 60 mm or more. The conical socket 11 projects for example up to a depth of 2 mm into the interchangeable head 1. Accordingly, the conical projection 21 projects approx. 2 mm beyond the end-side plane surface 25 of the shaft 2. The dimensions of the conical socket 11 and of the conical projection 21 are matched to one another, so that a predefined small air gap arises between the shaft-side plane surface 15 of the interchangeable head 1 and the end-side plane surface 25 of the shaft 2, with the loose assembly. These surfaces are pressed together on tightening the cap screw 3, and a controlled, predefined deformation of the conical projection 21 occurs on account of the predefined size of the air gap.

Figure 2 shows a longitudinal section through a head region of a shaft of a machine reaming tool according to the invention. The shaft or at least the head region is preferably designed as one piece. The axial bore 26 is designed in a continuous manner and at its tool-side end has a thread 23. The axial bore 26 in the region of the conical socket 11 has a diameter which is increased with respect to the threaded bore 23. On account of this, a free region for deformation of the conical socket 11 remains between the material of the conical socket 11 and the cap screw 3. The orientation pin 22 is pressed into a bore in the shaft 2. In principle, the orientation pin may be arranged on the exchangeable cutting tip 1, and the corresponding orientation hole on the shaft 2. This embodiment however is slightly more complicated in manufacture, particularly since the interchangeable heads 1 must be exchanged more often than shafts 2.

Figure 3 shows a shaft 2 according to the invention. The shaft 2 on the machine side, thus at its end which is distant to the tool 1, is envisaged for chucking into a machine. For this, it is for example of a circularly cylindrical shape or comprises standard chucking surfaces. Exemplary dimensions are a shaft length of 80 mm and a shaft diameter D2 of 10 mm with reaming head diameters of 11 mm to 16 mm, or a shaft length of 110 mm and a shaft diameter D2 of 16 mm with reaming head diameters of 18 mm to 24 mm. Preferably no guide portion for guiding the shaft in the bore hole is formed on the shaft itself.

Figures 4, 5 and 6 in each case show corresponding views of interchangeable heads according to various embodiments of the invention. Figure 4 shows an embodiment with a central cap screw 3 according to the Figures 1 to 3. A continuous recess 50, 13, 14 on the shaft side is formed as a cutout 50 or conical socket 11, on the oppositely lying side as a pocket for screw head 14, and therebetween as a bore 13, respectively.

In contrast to the Figures 1 and 7, in which the periphery of the interchangeable heads is only shown schematically according to the shape of the semi-finished product, the periphery is shown more accurately in the Figures 4, 5 and 6. The interchangeable head 1 comprises several cutting teeth with cutters 16, which are distributed in the peripheral direction. A cutter leads in the axial direction from a first to a second plane end-face of the interchangeable head 1. The end-faces delimit the interchangeable head 1 in the axial direction and run perpendicularly to the axial direction and at a distance h parallel to one another. A cutter 16 comprises a leading cut portion 17 and a guide portion 18. The guide portion is comparatively short so that the distance h is also comparatively short. Despite the very short guide portion 18, a useful application of the machine reaming tool according to the invention is possible in most cases of application.

Figure 5 shows a preferred embodiment of the invention, in which several continuous bores in each case with a pocket for a screw head 14, are arranged concentrically about a central bore 13. An adequate fastening and torque transmission is ensured by way of this, even with those annular interchangeable heads 1 with a larger diameter. Since with this embodiment, the pockets for the screw heads 14 and the connection elements such as conical socket 11 are arranged next to one another and not on the same axis, the maximal thickness h_1 may even be less than 3 mm or 2 mm.

Figure 6 shows a further preferred embodiment of the invention in which the conical socket 11 which comprises three contact segments 52 which are exposed, which means to say that the remaining regions of the conical socket have a slightly larger inner radius. The contact segments 52 are distributed distanced to one another in the peripheral direction and uniformly over the periphery. On manufacture, only the surfaces of these contact segments 52 may be manufactured or ground with a high accuracy.

Figures 7 and 8 show sections through an interchangeable head 1 in a further embodiment of the invention. In this, as also in Figure 9, the connection elements 27, 50 are shaped essentially cylindrically. Figure 7 shows a cross section parallel to the axial direction and Figure 8 a cut-out of a cross section perpendicular to the axial direction. A cutout 50 acting as a connection element is mainly shaped in a circularly cylindrical manner, but has three contact segments 51, 52. With each contact segment 51, 52, the inner wall of the circular cylinder is plane over a sector of the circle. An extension of such a sector corresponds roughly to a fifth to half the cylinder diameter. With a cylinder diameter of approx. 17 mm this for example corresponds to 2 mm to 4 mm. Preferably, one of the contact segments 52 is significantly larger in the peripheral direction, for example double as large as the others and therefore acts as a catch segment 51 by way of this.

Figure 9 shows corresponding sections through a shaft corresponding to the interchangeable head from the Figures 7 and 8. The connection element on the shaft 27 is

essentially circularly symmetrical, with a planarly machined sector, thus a plane surface on the shaft periphery 28. The connection element in the axial direction projects for example by approx. 1 to 2 or 4 mm beyond the end-side plane surface 25 of the shaft 2. The catch segment 51 and the plane surface on the shaft periphery 28 must be aligned to one another for the assembly of the interchangeable head 1 and the shaft 2.

Figure 10 shows a cut-out of connection elements of interchangeable head and shaft according to a further embodiment of the invention. The connection element on the shaft 27 and accordingly the cutout 50 on the interchangeable head 1, are shaped as a rounded triangular cylinder projection and socket respectively. The connection element on the shaft 27 is slightly smaller than the cutout 50, so that firstly a loose connection may be created. Figure 10 for illustration shows a greatly exaggerated size difference. On assembly or on operation of the machine reaming tool, contact points 53 arise due to the mutual rotation of the interchangeable head 1 and the shaft 2.

Figure 11 shows an interchangeable head designed as a reversible insert, in a partly cut-open lateral view and a detailed view Y. The guide portion 18 has a first taper 181 and a second taper 182, which in each case lead to the middle of the cutter 16 from the first reaming corner 191 and from the second reaming corner 192 respectively. The diameter of the edge of the cutter 16 thereby reduces towards the middle, wherein the angle α_1 between the edge and a connection line between the first reaming corner 191 and the second reaming corner 192 is a few hundredths of a degree. This corresponds to a diameter change of about 0.01 mm per 10 mm in the axial direction.

A groove 183 is preferably arranged between the two tapers 181, 182. This simplifies the manufacture, for example the grinding of the surfaces corresponding to the tapers 181, 182. A length of the tapers 181, 182 between the groove 183 and the reaming corners 191, 192 is for example 0.5 or 1 mm in each case, which is shown in **Figure 11** by the lengths V1 and V2.

A first leading cut portion 171 leads from a first reaming corner 191 to a first plane surface 193, and from the second reaming corner 192, a second leading cut portion 172 leads to a second plane surface 194. The two leading cut portions 171, 172 are provided for cutting, they thus have a back-off clearance, or clearance angle or clearance relief 173 which means that the radius of the cutting tooth 16 along the periphery reduces behind the cutting edge. Ground surfaces which form the clearance relief 173 are shown in **Figure 15**. The leading cut portions 171, 172 typically have an angle of 45° to the plane surfaces 193, 194, but may also be inclined to a greater or lesser extent.

Figure 12 shows a lateral view in cross section and a plan view of a reversible insert according to Figure 11. The cutters 16 therein are represented in a simplified manner. The

reversible interchangeable head 1 comprises first connection means 197 and second connection means 198, wherein each of these connection means consists of a set of through-holes with sinkings 14 for accommodating a screw head. These sinkings 14 with the first set of holes are arranged on the first plane surface 193, and with the second set of holes are arranged on the second plane surface 194.

A first conical socket 195 leads from the first plane surface 193 and a second conical socket 196 leads from the second plane surface 194, coaxially to the axis of the interchangeable head 1 into the interchangeable head 1, for centring the interchangeable head 1. In a first fastening position of the interchangeable head 1 on the shaft, the first plane surface 193 is the front one, and the second plane surface 194 and the first conical socket 195 are held or pressed against the shaft 2 by way of screws through the first set of holes 197. The interchangeable head 1 is turned over in a second fastening position, so that the second plane surface 194 is the front one, and the first plane surface 193 and the second conical socket 196 are pressed against the shaft 2.

On using the interchangeable head 1 in the first fastening position, the first reaming corners 191 of the cutters 16 are worn. What is surprising is the fact that with this, the subsequent second reaming corners 192 are not significantly worn. The interchangeable head 1 is turned over when the first reaming corners 191 have been worn to such an extent that the demanded dimension is no longer achieved after the reaming, typically by a few hundredths of a millimetre.

Figure 13 shows a pre-machining tool 200 for attachment on an interchangeable head. The premachining tool 200 in the following is indicated as a planisher 200. It has a slightly smaller premachining diameter D3 or preliminary dimension than the end dimension according to the interchangeable head diameter D1. The planisher 200 shown in the Figure comprises individual reversible inserts 305 for a machining with material removal, and is fastened to the same shaft 2 as the interchangeable head 1 by way of a shaft 201 of the planisher 200. In addition, the shaft 201 of the planisher 200 comprises two plane surfaces 203, and an axial sinking in the shaft 2 is shaped corresponding to this, for an improved torque transmission.

Figure 14 represents a shaft 2 with an interchangeable head 1 assembled thereon and a planisher 200. The interchangeable head 1 for this comprises a set of through holes for receiving sunk screws as in the **Figures 5 or 12**. The screws run essentially parallel to the axis of the shaft 2. The planisher 200 is held by way of a centric screw 204 and is tightened against the interchangeable head 1 in the axial direction. The screw 204 is led through a bore 202 in the planisher 200 and is screwed in the shaft 2.

Figure 15 shows an interchangeable head designed as a combined reaming head and miller, in a partly sectioned open view and plan view. As already described in the context of

Figure 11, here too the two leading cut portions 171, 172 of a cutter 16 or tooth 16 are provided for cutting or machining, and thus have a clearance relief or back-off clearance 173. In the embodiment of the invention according to Figure 15, the first leading cut portion 171 leads from the first plane surface 193 to the first reaming corner 191 and from there, the guide portion 18 leads with a slight tapering to an oppositely lying corner which in this case is indicated as a milling corner 199. The second leading cut portion 172 leads from the milling corner 199 to the second plane surface 194.

In contrast to the embodiment according to **Figure 4** where the edge between the cutter 16 and the shaft-side plane surface 15 is only broken by a chamfer, here the edge arising by way of the chamfering is designed as a cutting leading cut edge 172 with a back-off clearance 173. The cutter 16 is formed on a first as well as second side of the interchangeable head 1. One may for example mill a chamfer on the edge of the exit hole with the second leading cut portion 172 for example after the reaming of a bore and the exit of the interchangeable head 1 from the bore. For this, the axis of the rotating interchangeable head 1 is moved on a circular path about the axis of the bore by way of interpolating milling or circular milling.

Figure 15 in two sections shows a further preferred design of the two leading cut portions 171, 172. The section A-A shows a view of a first leading cut portion 171 in front of a reaming corner 191. In order to ensure a high mechanical strength of the first leading cut portion 171, it has a relatively small backslope angle β_1 of approx. 3° to 10° , preferably at least approximately 6° . The angle β_1 is measured with respect to a tangent to the leading cut portion 171. Preferably the back-off clearance 173 after a distance b to the cutting edge has a larger angle β_2 , for example of 10° to 20° , preferably at least approximately 12° . The distance b is 0.05 mm or 0.1 mm to 1 mm, preferably at least approximately 0.25 mm, depending on the size of the interchangeable head 1.

The section B-B shows a view of a second leading cut portion 172 in front of the milling corner 199. Since no great demands are placed on the precision of the miller as on the reaming function, a single back-off clearance surface 173 with a backslope angle β_3 of 10° to 20° , preferably at least approximately 12° is sufficient.

Figures 16 and 17 in each case show corresponding views of semi-finished products 6 for interchangeable heads according to the invention. In particular the connection elements, which means to say the first and second plane surface 193, 194, the conical socket 11 and the through holes with an optional pocket 14 for a screw head are completed and machined on the semi finished products 6. Only the cutters 16 need yet to be ground according to the specific requirements. The two **Figures 16 and 17** show semi-finished products 6 for different diameters and with the same thickness of approx. 4.3 mm with different representational scales. The semi-finished product 6 of **Figure 16** has a diameter of 16 mm, that of **Figure 17** a diameter of 101

mm. Preferably however several interchangeable heads 1 with different diameters but with the same thickness h_1 form a set of interchangeable heads 1. The thickness h_1 preferably lies between 4 mm and 5 mm for all described embodiments.

5

LIST OF REFERENCE NUMERALS

1	reaming head, interchangeable head	6	semi finished product
11	conical socket	171	first leading cut portion
12	orientation hole	172	second leading cut portion
13	bore	173	back-off clearance
14	pocket for screw head	181	first taper
15	shaft-side plane surface	182	second taper
16	cutter	183	groove
17	leading cut portion	191	first reaming corner
18	guide portion	192	second reaming corner
2	shaft	193	first plane surface
21	conical projection	194	second plane surface
22	orientation pin	195	first conical socket
23	threaded bore	196	second conical socket
24	coolant channel	197	first connection means
25	end-side plane surface	198	second connection means
26	axial bore	199	milling corner
27	connection element on the shaft	200	planisher
28	plane surface on the shaft periphery	201	shaft of the planisher
3	head screw	202	bore
50	cutout	203	plane surface
51	catch segment	204	screw
51	contact segment	205	reversible insert
53	contact point		